



Optimum Choice of Refrigerant for Miniature Vapour Compression Refrigeration System

KEYWORDS

Miniature refrigerator, electronic cooling, R134a, COP.

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ABSTRACT Recently in most of the studies mesoscale vapour compression refrigerator is integrated in high power microelectronic packaging and it offers a new cooling solution. Thermodynamic efficiency of such system depends on its operating temperatures. However, important practical issues like system design, sizing, cost, safety, reliability etc. depends on the type of refrigerant selected for given application. Now a days, global warming and ozone layer depletion are the prior issues to be considered while selecting the refrigerant. This work presents a comparison between four refrigerants and finding out the most suitable one for miniature refrigerator. In this work it is found that NH₃ gives the highest COP but because of its drawbacks like corrosive nature, fouling smell and highly toxic nature R134a is selected for the system.

Nomenclature

Q Heat (W)
W Power input (W)
h Specific enthalpy (kJ/kg)
T Temperature (°C)
COP Coefficient of performance
RE Refrigerating effect

Greek Symbols

η efficiency

Subscripts

C compressor
c condenser
e evaporator

Introduction

In recent years the development of power electronic devices [1] has led to many thermal issues. Air cooling technology has been in use for cooling of electronic devices since many years. But now a days liquid cooling is also attracting increasing interest as it is more effective than air cooling so

it becomes necessary to develop a new technique which will serve the purpose. From the study of various alternatives such as Heat pipe, Liquid immersion, Thermoelectric, VCR and Jet impingement [2], the vapour compression system is found to be most effective for dissipating heat[3,4,5].

For developing such a small size refrigerator it is necessary to select the type of refrigerant to be used before designing the system [6]. There is no general rule governing the selection of refrigerants. There are of course five basic criteria thermo physical properties, technical aspects, economic aspects, safety and environment factors.

Increasing concern about climate change has resulted in the developments of efficient and energy conserving refrigerants. New refrigerants are developed keeping in mind its effect on Ozone Depletion potential (ODP) and Global Warming Potential (GWP) [7]. Considering the contribution to climate change it is important to select the refrigerant considering their performance merits and properties. Hence in this investigation a comparative study of four different refrigerants is done. Table-1 shows different properties and applications of refrigerants.

Table-1 Properties and applications of various refrigerants.

Sr. No.	Refrigerant	Boiling Point	Critical Temperature	Properties	Applications
1	Ammonia (NH ₃)-R717	-33°C	133°C	Penetrating smell, Non-flammable but Explosive, efficient, Attacks copper, No ODP and GWP, Highly toxic.	Large industrial plants. Since it attacks copper, used in open type reciprocating or screw compressors and steel pipes
2	R12 (CCl ₂ F ₂)	-30°C	112°C	Has little odour, Colourless, Non-flammable, stable and Noncorrosive, Has ODP and GWP	Small plants with reciprocating compressors, used in domestic and commercial refrigeration.
3	R22 (CHClF ₂)	-40.8 °C	96°C	Colourless, Non-flammable, Noncorrosive and stable, Has ODP and GWP	Packaged air conditioning unit where size and economy of equipment are important.
4	R134a (CH ₂ FCF ₃)	-26°C		Colourless, Non-flammable, Noncorrosive Nontoxic and stable, GWP-1300	Centrifugal and high speed screw compressor, widely used automobile A/C.

Effect of refrigerant type on COP

Fig. 1 Simple Vapour compression cycle.

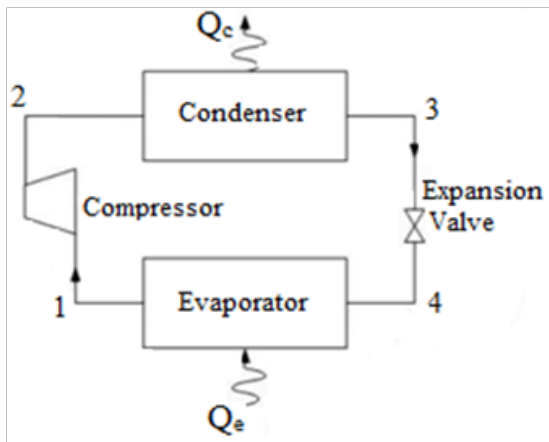


Fig. 1 shows a simple vapour compression cycle and component details. It consists of a compressor, condenser, expansion valve and evaporator. Evaporator absorbs the heat from the Integrated Circuit (IC) chip connected to it. Heat dissipated by chip is given by Q_e and heat rejected by condenser is shown by Q_c . WC is the power input given to the compressor in form of electric supply. Work is done on compressor while compressing the refrigerant in the process (1-2). Hence net work done during the cycle equals to compressor work i.e.

$$WC = (h_2 - h_3) \text{ kJ/kg} \quad (1)$$

Heat is rejected at the condenser end during process (2 - 3) i.e.

$$Q_c = (h_2 - h_3) \text{ kJ/kg} \quad (2)$$

Refrigerating effect is the heat absorbed by the vaporizing refrigerant in the evaporator during process (4 - 1) i.e.

$$R.E. = Q_e = (h_1 - h_4) \quad (3)$$

The efficiency of refrigeration system is given by COP.

$$COP = Q_e / WC = (h_1 - h_4) / (h_2 - h_1) \quad (4)$$

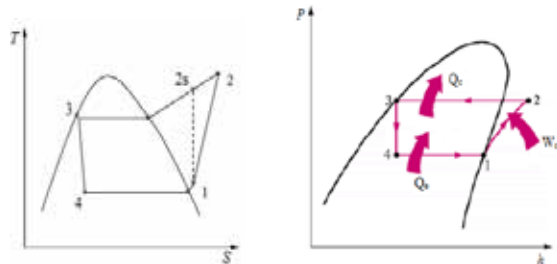


Fig. 2 P-h and T-S diagrams of Vapour compression cycle.

In ideal process compressor work is assumed as isentropic but in actual practice it is not so. The actual work is more than the isentropic work as shown in the Fig by line (1-2). So a term isentropic efficiency come into account and given by,

$$\eta_C = \frac{(h_{2s} - h_1)}{(h_2 - h_1)} \quad (5)$$

Table-2 summarizes our calculated results using software named COOLPACK [8] and shows the effect of T_e and η_C on COP of four refrigerants (NH3, R12, R22, and R134a).

$$COP = \frac{T_e}{(T_c - T_e)} \quad (6)$$

From equation (6) it is clear that as the difference between T_c and T_e decreases the value of COP increases. Same trend can be observed in the Table-2. It is clear from Table-2 that for same values of T_e , T_c and η_C , NH3 yields highest value of COP followed by R12, R134a and R22. Apart from this NH3 possesses very large latent heat $h_{fg}=1225.039$ kJ/kg then R22 possesses $h_{fg}=196.958$ kJ/kg after that R134a possesses $h_{fg} = 190.87$ kJ/kg and at last R12 possesses $h_{fg}= 146.363$ kJ/kg, at 100C.

Table-2 Effect of refrigerants on COP.

T_c (°C)	T_e (°C)	η_C	COP			
			NH3	R12	R22	R134a
-10	50	0.85	2.88	2.85	2.78	2.76
-10	50	0.75	2.54	2.52	2.45	2.44
-10	50	0.6	2.03	2.01	1.96	1.95
-5	50	0.85	3.27	3.24	3.15	3.15
-5	50	0.75	2.88	2.86	2.78	2.78
-5	50	0.60	2.31	2.29	2.22	2.22
0	50	0.85	3.74	3.71	3.60	3.62
0	50	0.75	3.30	3.28	3.18	3.20
0	50	0.60	2.64	2.62	2.54	2.56
5	50	0.85	4.31	4.29	4.16	4.20
5	50	0.75	3.81	3.79	3.67	3.71
5	50	0.60	3.04	3.03	2.94	2.96
10	50	0.85	5.03	5.02	4.86	4.93
10	50	0.75	4.44	4.43	4.28	4.35
10	50	0.60	3.55	3.54	3.43	3.48
15	50	0.85	5.96	5.95	5.75	5.86
15	50	0.75	5.26	5.25	5.08	5.17
15	50	0.60	4.21	4.20	4.06	4.14

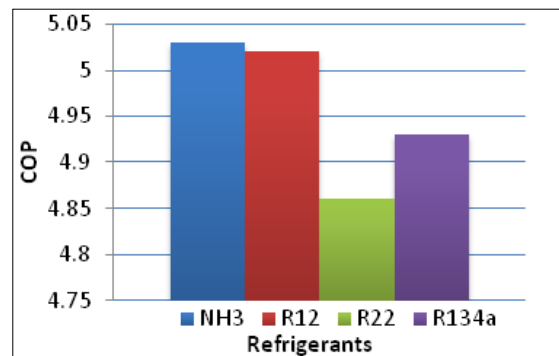


Fig. 3 COP of different refrigerants for $(T_c - T_e) = 400C$ and $\eta_C = 0.85$

Conclusion

A comparison of refrigerants NH3, R12, R22 and R134a reveals that NH3 gives the maximum value of COP. But NH3 cannot be used because of its drawbacks such as fouling smell and toxic nature. It is non-flammable but explosive. Next refrigerant is R12 having highest COP after NH3. R12 is basically stable and non-flammable but it causes serious damage to ozone layer. Hence R12 cannot be used. After that R22 also have same drawbacks as that of R12. It also causes harmful effects to ozone layer.

Now last refrigerant is R134a which is most widely used because of its certain advantages over the others like non-toxic, non-flammable, high COP and also harmless for Ozone layer. Also existing compressors are designed for R134a. Hence, it can be said that R134a is the most suitable refrigerant for miniature refrigerator.

Future Scope: - Experimentally investigate the developed miniature refrigerator using the above mentioned refrigerants.

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