



Comparison of Performance and Emission Characteristics of Rapeseed Biodiesel and Diesel

KEYWORDS

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ABSTRACT *The present study is carried on the comparison of performance and emission output of a compression ignition (CI) engine by using Rapeseed Methyl Ester (RME) and diesel. The experiments are to be proceed with rapeseed biodiesel that is prepared by transesterification process and diesel. The experiments starts with the engine is operated at low load and then keep on increasing till full load and variable engine speed conditions and the power and brake specific fuel consumption (BSFC) are calculated. At 4 to 15bar the emission and performance test of Brake Mean Effective Pressure (BMEP) at 1500 rpm engine speed which this condition is predefined from first stage of the 15 experiments. CO, CO₂ and NO_x values are measured for two different fuel with varying load. Rapeseed biodiesel results ~13% decrease at Brake Effective Power (BEP) and 9% increase on BSFC. The NO_x emissions are decreased 40% when fueling with biodiesel while there were significant decrease was noted for CO and CO₂ emission by using the system as the same condition used for performance.*

1. Introduction

Energy is the most valuable and important requirement of modern life. Primary energy sources of the world such as petrol are finite sources and from the early stages of the industrialization total production of oil has increased.[1] The increasing energy consumption has created a demand for sustainable and clean energy sources. This situation encouraged the scientists to search for alternative renewable and sustainable fuel instead of fossil based fuels. Internal combustion engines are the main energy conversion machines and have widespread use because of its' compact structure, high thermal efficiency and proved technology. Vegetable oils and derivatives are one of the alternative fuel that can be replaced with conventional diesel fuel considering to the internal combustion engines.

Vegetable oils have higher density, viscosity and volatility and have bigger molecular structure than diesel fuel[2], these different properties negatively affect injection and spray atomization of fuel in combustion chamber. Poor atomization of vegetable oils cause higher deposit formation on combustion chamber and injector nozzles that reduces engine operation period when compared to mineral diesel fuel.[3,4] Vegetable oils can be used as fuel with additional methods which are blending with mineral diesel, heating of oil for reducing viscosity and converting to fatty acid esters with applying transesterification. Fatty acid methyl esters (FAME) have similar properties when compared to petroleum diesel, thus implementation as fuel or additive can be feasible.[4]

Hemmerlein et. al. compared the performance and emission characteristic of rapeseed oil with diesel oil at several different engine types. Engines which investigated in this study were covered the range from 40 to 274 kW power output. Torque and power output with rapeseed oil were only 2% lower than with Diesel fuel when lower heating value of rapeseed oil were 7% lower. Authors explained the situation with higher injected fuel quantity of rapeseed oil than with Diesel fuel depending on higher viscosity of rapeseed oil. Carbon monoxide emissions are within the whole engine operating range up to 100 % higher and hydrocarbon emissions identically rised with rapeseed oil compared to Diesel fuel. Emissions of nitrogen oxides

were up to 25 % lower with rapeseed oil. Authors related results with slower combustion and lower maximum temperature in the combustion chamber.

FAME have similar fuel properties with Diesel fuel, also it is biodegradable and renewable source of energy. FAME so called biodiesel also has higher viscosity, higher density and higher bulk modulus than diesel fuel[2,6], which expose quite different injection characteristics (injection pressure, injection timing, spray tip penetration, cone angle etc.)[7,8,9] than those of mineral Diesel fuels that leads to difference in tail pipe emissions, deposit formation and performance of the engine.[10]

When using biodiesel instead of diesel fuel in a compression ignition (CI) engine, most common and the significant effects on gaseous emissions and performance of engine are well studied by researchers[11,12,13,14]. Generally higher fuel consumption and lower power output obtained with biodiesel which it is related to biodiesel's lower heating value. Lapuerta et. al. pointed out the Nitric Oxide (NO_x) emissions are tends to increase because of advancing phenomenon of the injection start that originates from the physical properties of the biodiesel while the total hydrocarbon emissions are tends decrease because of the oxygen content of biodiesel. The general trend for Carbon monoxide (CO) emissions are towards to reduce because of oxygen enrichment that comes from the fuel, helps for a more complete combustion. At the other hand from the perspective of Carbon dioxide (CO₂) greenhouse gas, results are similar.

Labeckas et.al. investigated the effect of Rapeseed Methyl Ester (RME) and blends with diesel fuel on direct injection diesel engine performance and tail pipe emissions. Results of author's experiments indicate, Brake Specific Fuel Consumption (BSFC) of neat RME was 18% and 23% higher than diesel oil, also at fully opened rack position and low speed, the BSFC was lower by 3.2% and 1.7% for the B10 and B20 blends. At higher engine speeds, BSFC was lower by 3.5% for both the B5 and B10 blends, at rated power, the B5 blend suggests slightly better (1.5%) fuel economy.

A modern internal combustion engine has to meet several demand which mandated by regulations and consumers. Environment pollution related to engines become a major problem thus harmful combustion products has to be reduced.

The purpose of this research was to investigate the effect of biodiesel on internal combustion engine performance, exhaust emissions. Several tests carried on a diesel engine

and results summarized.

2. Materials and methods

2.1 Fuels

Test fuels are RME and Diesel fuel. Two different type of fuels used are RME and diesel. Fuel properties are listed in table 1 and table 2.

Table 1 Specifications of Diesel fuel

Specification	Unit	Test method	Limit			Test
			Min	max	results	
Specific gravity	kg/m ³	ASTM D	820	860		850
15°C			4052			
Flash Point	°C	ASTM D 93	55			67
Water content	mg/kg	ASTM D		200		97.5
			6304			
Sulphur content	mg/kg	ASTM D		7000		1470
			2622			
Copper strip	3h, 50	ASTM D	Class 1			1A
corrosion	°C		130			
Cetane index	Calc.	ASTM D	46			52
			4737			
Viscosity 40°C	mm ² /s	ASTM D	2.0	4,5		2.812
			445			
Cold filter	°C		IP309			-20
plugging point						

Table 2 Specifications of Rapeseed methyl ester

Specification	Unit	Test method	Limit			Test
			min	ma	Results	
Specific gravity	kg/m ³	EN ISO	860	900		890
15°C			3675			
Flash Point	°C	EN ISO	2719	>101		>135
Water content	mg/kg	EN ISO	12937	-	500	475
Copper strip	3h, 50 °C	EN ISO		Class		1A
corrosion			1			
Cetane index	Calc.	EN ISO	5165	51	-	57
Viscosity 40°C	mm ² /s	EN ISO		3.5	5	4.7
			3104			
Acid value	mg KOH/g	EN 14104		-	0,5	0.42
Iodine value		EN 14111		-	120	102
FAME content	% (m/m)	EN 14103		96.5	-	97.2

2.2 Test bed and instruments

A single cylinder engine is used for performance and emission tests. A hydrokinetic dynamometer is used for loading and unloading the engine. The engine speed is measured by use of infrared tachometer. The engine torque is measured with load cell. Fuel consumption is determined

with volumetric method. The duration of 50 cm³ of fuel consumption is measured, at predefined engine operating point. Air consumption of the engine is measured with an inclined manometer via standard orifice which was mounted inlet side of an air oscillation absorber tank. A Bilsa brand Mod 2210 model gas analyzer was used to

determine CO, CO₂, NO_x emissions. Oil temperature was measured with using thermocouple which mounted on oil pan drain screw. During the tests, first the engine was set to desired load and speed condition then the measurements were taken. Scheme of test bench shown in figure 1 and details of test engine are listed in table 3.

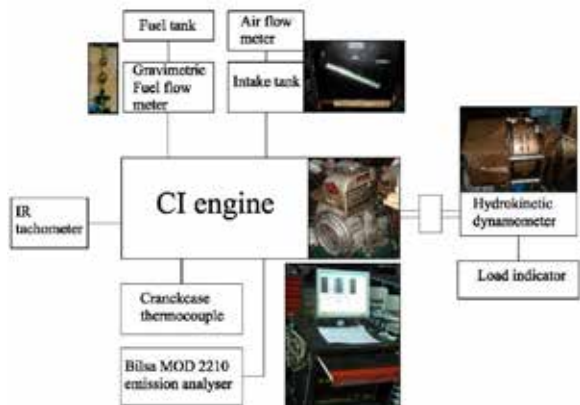


Figure 1- Schematic of test bench.

Table 3 Specifications of Engine Test Bed

Engine Manufacturer	Petter Diesel
Model-Type	AA1
Aspiration	Naturally aspirated
Number of cylinders	1
Cooling method	Air cooled
Combustion technology	Indirect injection-Turbulence chamber
Bore x Stroke (mm)	69.85x57.15
Cylinder volume (cm ³)	219
Compression ratio	17:1
Lubricating oil capacity (l)	1.9
Speed range Min-Max (rpm)	1500-3600
Fuel injection pressure (bar)	183
Fuel injection timing BTDC	33° CA
Rated power DIN ISO 3046 (kW)	2.5

2.3 Methodology

Several tests performed for determination of the effect of RME on engine performance and emissions. For measuring the brake power of test engine, fuel pump rack adjusted to maximum position and engine power were noted at 500 rpm speed intervals. Performance tests repeated ten times for each fuel and mean values taken into account for plotting the power curve of engine. Maintaining equal conditions is necessary for emission investigation, such as equal load at same engine speed or injecting an equal fuel quantity into cylinder. Because of the characteristic of in-line-pump system and governor speed limiter, it's impossible to inject equal quantity of fuel into cylinder for both type of fuels. RME has higher density and viscosity than diesel fuel, which these properties were the main reason of altering quantity of injected fuel per cycle. In order to obtain comparable results, equal load at equal speed comparison method was selected, load condition and engine speed determined as 4,15 Bars of Brake Mean Effective Pressure (BMEP) and 3050 rpm respectively. After setting engine to related test conditions, gaseous emissions were measured.

3. Results and discussion

3.1 Performance tests result

Results of brake power measurements for diesel fuel and RME are shown in figure 2. Brake effective power of RME was decreased 13% when compared to diesel fuel. Also maximum brake power speed of fuels are different, this can be related to different lower heating value and density of RME. [15]

According to figure 3 BSFC has increased when using RME as test fuel. Up to 3000 rpm of engine speed approximately 9% increase obtained for RME but BSFC values almost identical for higher engine speeds. Difference between BSFC increase and power decrease, indicates better combustion of biodiesel. Pump-line-nozzle fuel injection system delivers higher quantity of biodiesel at same conditions due to higher viscosity of fuel. In addition, the higher viscosity reduces leakages in the pump leading to an increase in the injection line pressure. Therefore, a quicker and earlier needle opening is observed when compared to diesel fuel. Higher injection pressure improves the efficiency of combustion also early injection provides longer time for fuel to burn completely.

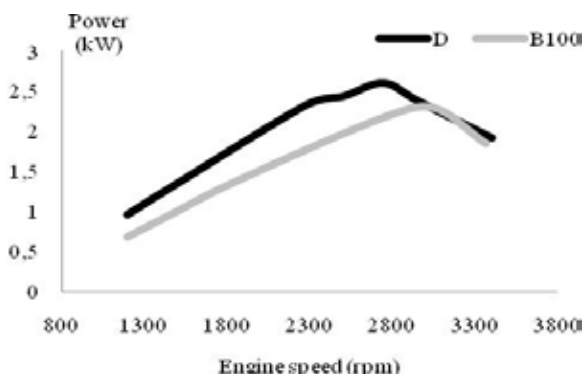


Figure 2. Brake Power as a function of engine speed.

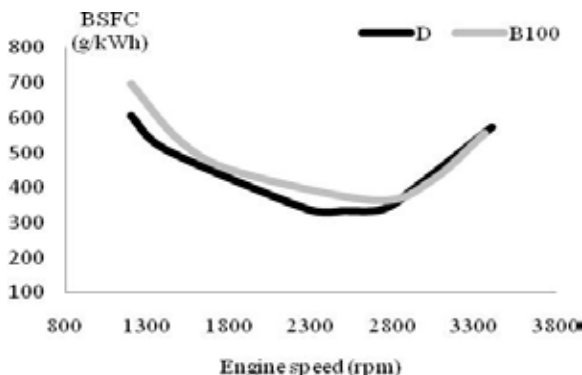


Figure 3. BSFC as a function of engine speed.

3.2 Emission test results

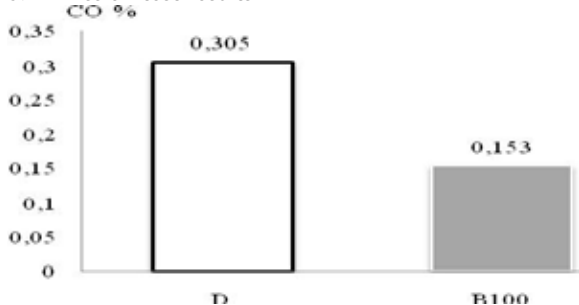


Figure 4. CO variation with respect to fuel type

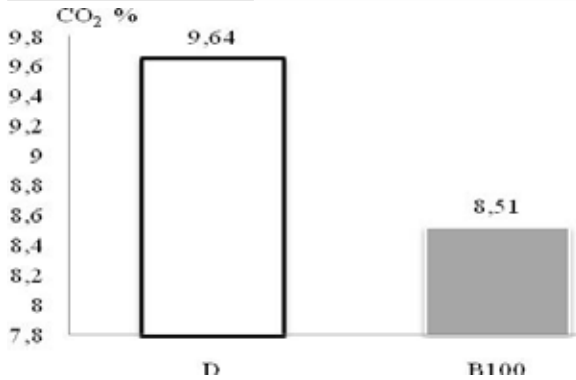


Figure 5. CO₂ variation with respect to fuel type

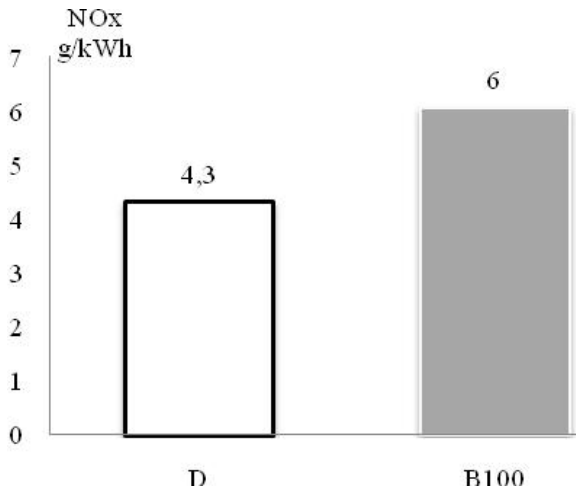


Figure 6. NO_x variation with respect to fuel type

Figure 5 shows CO variation at same operating condition, a significant reduction of CO emission obtained with RME. Higher cetane number of RME causes higher amount of fuel burnt in premixed combustion phase thus carbon related pollutants decreased considerably. Same trend also shown in figure 6 for CO₂ emissions of RME.

Major contributor of Nitric oxide emission is oxygen in hot burning zone. Higher amount of Nitric oxide can be expected with combustion of oxygenated fuels as like biodiesels. Nitric oxide measurements of RME are shown in figure 6, almost 40% increase obtained with biodiesel. The higher cetane number of biodiesel fuel as compared to diesel fuel can explain the above mentioned difference of biodiesel on NO_x emissions.

4. Conclusion

Biodiesel can be a fuel alternative for CI engines with its environment friendly production process. Also lower pollutant level of carbon related compounds are possible with using biodiesel. Although Nitric oxide emissions tend to increase with biodiesel.

Brake effective power of test engine decreased 13% and BSFC increased 9% with neat RME when compared to diesel fuel. Rapeseed oil methyl ester exposed lower carbon monoxide and carbon dioxide results in this study, which were decreased 50% and 12% respectively but 40% increase of NO_x emission obtained.

Nomenclature

FAME	Fatty acid methyl ester
RME	Rapeseed methyl ester
BSFC	Brake specific fuel consumption
BMEP	Brake mean effective pressure
NO _x	Nitric oxide
CO	Carbon Monoxide
CO ₂	Carbon Dioxide
B5	Blend with 5%vol. biodiesel with 95%vol. diesel fuel.
B10	Blend with 10%vol. biodiesel with 90%vol. diesel fuel.
B20	Blend with 20%vol. biodiesel with 80%vol. diesel fuel.
B100	Neat biodiesel

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